



Introduction to Marine Hydrodynamics (NA235)

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- The assignment can be downloaded from following website:
 Website: ftp://public.sjtu.edu.cn
 Username: dcwan
 Password: 2015mhydro
 Directory: IntroMHydro2015-Assignments
- Eight problems
- Submit the assignment on <u>March 19th</u> (in English, written on paper)





Two ways of describing a fluid flow:

Lagrangian description, Eulerian description



Eulerian description; Cartesian grid



Differences:

- Particle positions x, y, z in Lagrangian description; while spatial point (x, y, z) in Eulerian description.
- Spatial point (x, y, z) in Eulerian description is
- **independent** variable of *t*, however, particle positions *x*,
- y, z in Lagrangian description are **functions** of t.





Total (material, substantial) derivative:



Effects of the unsteadiness of the flow

Effects of the fluid particle moving (advecting or convecting) to a new location in the flow, where the velocity field is different



2.2 Pathlines and Streamlines

Various ways to visualize flow fields —

Pathlines





As pathlines are the <u>actual paths</u> traveled by individual fluid particles over some time period. In Lagrangian description, a pathline is the same as the fluid particle's position vector:

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$$\begin{cases} x = x(a,b,c,t) \\ y = y(a,b,c,t) \\ z = z(a,b,c,t) \end{cases}$$

















A fluid particle moves from one spatial position (x, y, z) to another (x+udt, y+vdt, z+wdt) over a period of time dt, i.e., it moves a distance of dr. The equation of the **pathline** for the fluid particle is:



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Since the pathline is defined by integration of the relationship between velocity and displacement, to integrate (u, v, w) with respect to t, use initial condition (x_0, y_0, z_0, t_0) to determine the integral constants, then eliminate t.







Definition: a streamline is a curve that is everywhere tangent to the velocity vector at a given instant.

At an instant of time, there is at every point a velocity vector with a definite direction. The instantaneous curves that are everywhere tangent to the direction field are called the streamlines of flow.

Two-dimensional flows

By definition, the local velocity vector \mathbf{V} and the element of arc length along a streamline d**r** are locally parallel, thus the equation for a streamline is:

$$\mathbf{V} \times d\mathbf{r} = \mathbf{0}$$
$$\underbrace{\frac{dx}{u} = \frac{dy}{v} = \frac{dz}{w}}_{w}$$

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Also by simple geometric arguments using <u>similar triangles</u>, the slope of the streamline:

$$\frac{dy}{dx} = \frac{v}{u}$$







Pathlines and streamlines are identical in steady flows





Characteristics of streamlines:

- Instantaneous quantities
- ◆ Tangential direction of the streamlines are identical to the velocity vectors. Streamline cluster <u>density</u> reflects the <u>magnitude</u> of velocity: streamlines close together ⇒ high velocity, streamline far apart ⇒ low velocity
- The streamlines never intersect each other except at a point of zero velocity, because at any point, there can be only one direction of the velocity
- The streamlines **never interrupt** in the fluid
- Since the velocity vector in the flow field is everywhere tangent to the streamline, the fluid cannot cross the streamlines and the streamlines can be regarded as <u>fixed</u> walls



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Definition: a set of streamlines that intersect a closed loop in space.

Streamtubes are instantaneous quantities like streamlines, defined at a particular instant in time according to the velocity field at that instant. When the flow is steady, the shape of the streamtube does not change with t, like a real pipe. **No** fluid crosses a stream tube's surface because the fluid velocity vector is everywhere tangent to it.





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<u>Streamtube</u>

Let's take **two cross-sections** of a streamtube, with cross-sectional areas A_1 and A_2 . The velocities **perpendicular** to the cross-sections are V_1 and V_2 , respectively. The rate at which **mass** is <u>entering</u> the streamtube is $\rho_1 A_1 V_1$; the rate at which it is <u>leaving</u> is $\rho_2 A_2 V_2$. If the flow is **steady**, the mass is conserved and then $\rho_1 A_1 V_{1=}$ $\rho_2 A_2 V_2$, and if flow is **incompressible**, then $A_1 V_{1=} A_2 V_2$.

The fluid speed increases when the crosssectional area of the streamtube decreases.





In an incompressible flow field, a streamtube (a) **decreases** in diameter as the flow **accelerates** or <u>converges</u> and (b) **increases** in diameter as the flow **decelerates** or <u>diverges</u>.



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Definition: the flow rate of a property (volume, mass, weight) per unit area of a spatial curved surface per unit of time — volumetric flux, mass flux, weight flux.



For a closed surface S, take the unit outer normal as positive, then the flux is:

$$\oint_{S} (\rho \mathbf{V} \cdot \mathbf{n}) dS = \iiint_{\Omega} \nabla \cdot (\rho \mathbf{V}) d\Omega \blacktriangleleft$$

 $\vec{V} \cdot \vec{n} = |\vec{V}|| \vec{n} | \cos \theta = V \cos \theta$ If $\theta < 90^\circ$, then $\cos \theta > 0$ (outflow). If $\theta > 90^\circ$, then $\cos \theta < 0$ (inflow). If $\theta = 90^\circ$, then $\cos \theta = 0$ (no flow).

Gauss' theorem

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A two-dimensional velocity field is given as: $u = \frac{x}{1+t}$, v = y(1) Determine the pathline equation for $(x)_{y=1,t=0} = 1$

(2) Determine the streamline equation for $(x)_{t=0} = a$, $(y)_{t=0} = b$

Solution: (1)

The pathline equation is: $\frac{dx}{dt} = \frac{x}{1+t}, \quad \frac{dy}{dt} = y$

Integrating to give: $\ln x = \ln(t+1) + \ln C_1$, $\ln y = t + \ln C_2$

i.e.,
$$x = C_1(1+t), y = C_2e^t$$

From $(x)_{y=1, t=0} = 1$: $C_1 = 1$, $C_2 = 1$

Thus, the pathline equation is:

$$\begin{cases} x = 1 + t \\ y = e^t \end{cases}$$

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(2) The streamline equation is:
$$\frac{1+t}{x} dx = \frac{1}{y} dy$$

Integrating, yield: $(1+t)\ln x = \ln y + \ln C$

i.e.,
$$x^{1+t} = Cy$$

From
$$(x)_{t=0} = a$$
, $(y)_{t=0} = b$: $C = \frac{a}{b}$

Thus, the streamline equation at t=0 and (a, b) is: $y = \frac{b}{a}x$



Assume the pathline equation of a fluid particle is:

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$$x = C_1 e^t - t - 1$$

$$y = C_2 e^t + t - 1$$
 where C_1 , C_2 , C_3 are constant

$$z = C_3$$

Determine: (1) The pathline equation of the fluid particle at x=a, y=b, z=c, and t=0;

(2) The velocity of any fluid particle;

- (3) The expression of velocity field by Eularian description;
- (4) Are the acceleration field by Eularian description and the acceleration field by converting Lagrangian description to Eularian description the same?





Solution: (I) The pathline equation

Substituting t = 0, x = a, y = b, z = c into the pathline equation, yields:

$$a = C_1 - 1, \quad b = C_2 - 1, \quad c = C_3$$

Thus,
$$C_1 = a + 1$$
, $C_2 = b + 1$, $C_3 = c$

The pathline equation of the fluid particle at (a,b,c) is:

$$x = (a+1)e^{t} - t - 1$$

$$y = (b+1)e^{t} + t - 1$$
 (1)

$$z = c$$





(2) The velocity of any fluid particle:

$$u = \frac{\partial x}{\partial t} = C_1 e^t - 1 = (a+1)e^t - 1$$
$$v = \frac{\partial y}{\partial t} = C_2 e^t + 1 = (b+1)e^t + 1$$
$$w = \frac{\partial z}{\partial t} = 0$$
(2)

Application2

(3) Expressing the velocity field by Eularian description

a, b, c are solved from Equation (1):

$$a = \frac{1}{e^{t}}(x+t+1) - 1$$

$$b = \frac{1}{e^{t}}(y-t+1) - 1$$
 (3)

c = z

Substituting into Equation (2), yields:

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$$u = \frac{\partial x}{\partial t} = (a+1)e^{t} - 1 = x + t$$

$$v = \frac{\partial y}{\partial t} = (b+1)e^{t} + 1 = y - t + 2$$
 (4)

$$w = \frac{\partial z}{\partial t} = 0$$



(4) Determining the acceleration field by Eularian description

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$$a_{x} = \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} = x + t + 1$$

$$a_{y} = \frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} = y - t + 1$$

$$a_{z} = \frac{\partial w}{\partial t} + u \frac{\partial w}{\partial x} + v \frac{\partial w}{\partial y} + w \frac{\partial w}{\partial z} = 0$$

From Equation (1), the acceleration of the <u>particle</u> at (a, b, c) is:

$$a_{x} = \frac{\partial^{2} x}{\partial t^{2}} = (a+1)e^{t}$$

$$a_{y} = \frac{\partial^{2} y}{\partial t^{2}} = (b+1)e^{t}$$

$$a_{z} = \frac{\partial^{2} z}{\partial t^{2}} = 0$$
(5)



Substituting Equation (3) to (5), yields:

$$a_x = x + t + 1$$
$$a_y = y - t + 1$$
$$a_z = 0$$

By comparison, the two results are the same



In theoretical mechanics, the motion of a <u>rigid</u> body can be split into the translational motion and the rotational motion.

$$V = V_M + \omega \times r$$

- V_M Velocity of a reference point M
- Radius vector between a moving point and
the reference point
- *O* Angular velocity

In fluid mechanics, to study the motion of the fluid, take a fluid element from the flow field. A point in this fluid element is M(x, y, z), an adjacent point is M' (x+dx, y+dy, z+dz). Let the velocity at M is **V**, then the velocity at M' is:

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$$\mathbf{V}_{M} = \mathbf{V}_{M} + \frac{\partial \mathbf{V}_{M}}{\partial x} \delta x + \frac{\partial \mathbf{V}_{M}}{\partial y} \delta y + \frac{\partial \mathbf{V}_{M}}{\partial z} \delta z + \cdots$$

$$= \left(u + \frac{\partial u}{\partial x} dx + \frac{\partial u}{\partial y} dy + \frac{\partial u}{\partial z} dz\right)\mathbf{i} + (\cdots)\mathbf{j} + (\cdots)\mathbf{k}$$

$$= \left[u + \frac{\partial u}{\partial x} dx + \frac{1}{2} \left(\frac{\partial u}{\partial y} - \frac{\partial v}{\partial x}\right) dy + \frac{1}{2} \left(\frac{\partial u}{\partial z} - \frac{\partial w}{\partial x}\right) dz\right]$$

$$+ \frac{1}{2} \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x}\right) dy + \frac{1}{2} \left(\frac{\partial u}{\partial z} + \frac{\partial w}{\partial x}\right) dz\right]\mathbf{i} + (\cdots)\mathbf{j} + (\cdots)\mathbf{k}$$

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2.3 Deformation and Rotation of Fluid Elements

In the equation:

(1)
$$\varepsilon_{xx} = \frac{\partial u}{\partial x}$$
 ε_{yy} ε_{zz}
(2) $\omega_z = -\frac{1}{2}(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y})$
(3) $\omega_y = \frac{1}{2}(\frac{\partial u}{\partial z} - \frac{\partial w}{\partial x})$ ω_x
(4) $\varepsilon_{xy} = \frac{1}{2}(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x})$
(5) $\varepsilon_{xz} = \frac{1}{2}(\frac{\partial u}{\partial z} + \frac{\partial w}{\partial x})$ ε_{zy}



Physical meaning of each item in the equation

(1) Physical meaning of
$$\varepsilon_{xx} = \frac{\partial u}{\partial x}$$
, $\varepsilon_{yy} = \frac{\partial v}{\partial y}$, $\varepsilon_{zz} = \frac{\partial w}{\partial z}$

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2.3 Deformation and Rotation of Fluid Elements

Consider the linear strain in x-direction only

Change of length per unit length:

$$\varepsilon_x = \frac{\partial u}{\partial x} dx dt / dx = \frac{\partial u}{\partial x} dt$$

Rate of change of length per unit length:

$$\mathcal{E}_{xx} = \frac{\mathcal{E}_x}{dt} = \frac{\partial u}{\partial x}$$

It indicates the rate of <u>increase</u> or <u>decrease</u> in length of a fluid element in x-direction.

Similarly:
$$\varepsilon_{yy} = \frac{\partial v}{\partial y}, \quad \varepsilon_{zz} = \frac{\partial w}{\partial z}$$

 $\mathcal{E}_{xx}, \mathcal{E}_{yy}, \mathcal{E}_{zz}$ are called linear strain rate —





Apparently:
$$\varepsilon_{xx} + \varepsilon_{yy} + \varepsilon_{zz} = \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = \nabla \cdot \mathbf{V}$$

 $\nabla \cdot \mathbf{V}$ is the divergence of velocity, denoting the rate of change of volume of a fluid element per unit volume (volumetric strain/dilatation rate).

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(b)

Change of volume in *x*-direction:

$$\delta \Psi_{x} = \left[\left(u + \frac{\partial u}{\partial x} \delta x \right) \delta y \cdot \delta z \cdot \delta t \right] - \left(u \cdot \delta y \cdot \delta z \cdot \delta t \right) = \frac{\partial u}{\partial x} \delta x \cdot \delta y \cdot \delta z \cdot \delta t$$

Change of volume in *x*-direction per unit volume per unit time:

$$\frac{\delta \Psi_x}{\Psi \cdot \delta t} = \frac{\frac{\partial u}{\partial x} \cdot \delta x \cdot \delta y \cdot \delta z \cdot \delta t}{\delta x \cdot \delta y \cdot \delta z \cdot \delta t} = \frac{\partial u}{\partial x} \text{ as } \delta x, \ \delta t \to 0_{-}$$

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2.3 Deformation and Rotation of Fluid Elements

Similarly, in *y* and *z* directions:

$$\frac{\delta \Psi_{\overline{y}}}{\Psi \cdot \delta t} = \frac{\partial v}{\partial y}, \qquad \frac{\delta \Psi_{\overline{z}}}{\Psi \cdot \delta t} = \frac{\partial w}{\partial z} \quad \text{as } \delta y, \ \delta z, \ \delta t \to 0$$

Thus, the total change of volume per unit volume per unit time is:

$$\frac{\delta \Psi}{\Psi \cdot \delta t} = \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = \nabla \cdot V \quad \text{as } \delta x, \ \delta y, \ \delta z, \ \delta t \to 0$$

The divergence of velocity $\nabla \cdot \mathbf{V}$ can be used to express the volumetric strain/dilatation rate

$$\nabla \cdot \mathbf{V} = 0 \quad \Leftrightarrow \text{Incompressible flow}$$



 $\nabla \cdot V > 0$, indicates the fluid flows out of the element and is called a source flow

 $\nabla \cdot \mathbf{V} < \mathbf{0}$, indicates the fluid flows into the element and is called a sink flow

 $\nabla \cdot \mathbf{V} = \mathbf{0}$, indicates the velocity field of the incompressible fluid is without the source flow.



(2) Physical meaning of
$$\mathcal{E}_{xy}$$
, \mathcal{E}_{xz} , \mathcal{E}_{zy}

(3) Physical meaning of ω_x , ω_y , ω_z

For a fluid element (in the *x*–*y* plane), the change of line OA in a short time interval δt is:

$$\delta \alpha = tg\left(\delta \alpha\right) = \frac{\partial v}{\partial x} \delta x \delta t / \delta x = \frac{\partial v}{\partial x} \delta t$$

Similarly,

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The angular velocity of line OA is:

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$$\dot{\alpha} = \frac{d\alpha}{dt} = \frac{\partial v}{\partial x}$$
 as $\delta x, \ \delta t \to 0$

Similarly, the angular velocity of line OB is:

$$\dot{\beta} = \frac{d\beta}{dt} = \frac{\partial u}{\partial y}$$
 as $\delta y, \ \delta t \to 0$



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2.3 Deformation and Rotation of Fluid Elements

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$$\therefore \varepsilon_{xy} = \frac{1}{2} \left(\frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} \right) = \frac{1}{2} \left(\dot{\alpha} + \dot{\beta} \right)$$

It denotes the rate of angular deformation of a right angle in the fluid element, it is called shear strain rate. When $\varepsilon_{xy} > 0$, the angle decreases; In contrast, the angle increases.

Similarly:

$$\varepsilon_{yz} = \frac{1}{2} \left(\frac{\partial w}{\partial y} + \frac{\partial v}{\partial z} \right)$$

$$\varepsilon_{zx} = \frac{1}{2} \left(\frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} \right)$$



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2.3 Deformation and Rotation of Fluid Elements

$$\omega_{z} = \frac{1}{2} \left(\dot{\alpha} - \dot{\beta} \right) = \frac{1}{2} \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right)$$

It denotes the rate of rotation, angular velocity of the fluid element. When $\omega_z > 0$, the fluid element rotates in <u>counterclockwise</u> direction; in contrast, it rotates in <u>clockwise</u> direction.



Similarly:

$$\omega_x = \frac{1}{2} \left(\frac{\partial w}{\partial y} - \frac{\partial v}{\partial z} \right), \quad \omega_y = \frac{1}{2} \left(\frac{\partial u}{\partial z} - \frac{\partial w}{\partial x} \right)$$





shear rate tensor
$$\mathbf{E} = e_{ij} = \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right)$$

rate of rotation $\boldsymbol{\omega} = \omega_x \vec{i} + \omega_y \vec{j} + \omega_z \vec{k} = \frac{1}{2} \boldsymbol{\zeta}$ (vorticity)

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vorticity $\boldsymbol{\zeta} = \boldsymbol{\Omega} = \nabla \times \mathbf{V}$ (curl of velocity)

