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Class-4

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Finite Element Analysis of Solids and Fluids

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Eigenvalue problems

Introduction

An eigenvalue problem is defined to be one in which we seek the values of the parameter λ such that the equation

$$A(u) = \lambda B(u)$$

is satisfied for nontrivial values of u . Here A and B denote either matrix operators or differential operators, and values of λ for which Eq. is satisfied are called eigenvalues. For each value of λ there is a vector u , called an **eigenvector or eigenfunction**

e.g.
$$-\frac{d^2u}{dx^2} = \lambda u(x), \quad \text{with } A = -\frac{d^2}{dx^2}, B = 1$$

which arises in connection with **natural axial vibrations of a bar** or the **transverse vibration of a cable**, constitutes an eigenvalue problem. Here λ denotes the square of the frequency of vibration, ω

Eigenvalue problems

In general, the determination of the eigenvalues is of engineering as well as mathematical importance

- In **structural** problems, the eigenvalues denote either natural frequencies or buckling loads
- In **fluid mechanics** and heat transfer, eigenvalue problems arise in connection with the determination of the homogeneous parts of the transient solution
- Eigenvalues often denote amplitudes of Fourier components making up the solution
- Eigenvalues are also useful in determining the stability characteristics of temporal schemes

Eigenvalue problems

Formulation of Eigenvalue Problems

Parabolic Equation

Consider the partial differential equation

$$\rho c A \frac{\partial u}{\partial t} - \frac{\partial}{\partial x} \left(k A \frac{\partial u}{\partial x} \right) = q(x, t)$$

which arises in connection with transient heat transfer in one-dimensional systems (e.g a plane wall or a fin). u denotes the temperature, k the thermal conductivity, ρ the density, A the cross-sectional area, c the specific heat, q the heat generation per unit length

- Equations involving the **first-order time derivative** are called parabolic equations

Eigenvalue problems

The homogenous solution (i.e, the solution when $q = 0$) is often sought in the form of a product of a function of x and a function of t (i. e, through the **separation-of-variables technique**)

$$u^h(x, t) = U(x)T(t)$$

Substitution of this assumed form of solution into the homogeneous form gives

$$\rho cA \frac{\partial u}{\partial t} - \frac{\partial}{\partial x} \left(kA \frac{\partial u}{\partial x} \right) = q(x, t)$$

Separating variables of t and x (assuming that ρcA and kA are functions of x only), we arrive at

$$\frac{1}{T} \frac{dT}{dt} = \frac{1}{\rho cA} \frac{1}{U} \frac{d}{dx} \left(kA \frac{dU}{dx} \right)$$

Note that the left-hand side of this equation is a function of t only while the right-hand side is a function of x only

Eigenvalue problems

For two functions of two independent variables to be equal for all values of the independent variables, both functions must be equal to the same constant, say $-\lambda$ ($\lambda > 0$):

$$\frac{1}{T} \frac{dT}{dt} = \frac{1}{\rho c A U} \frac{d}{dx} \left(k A \frac{\partial u}{\partial t} \right) = -\lambda$$

$$\frac{dT}{dt} = -\lambda T \quad (\text{p-1})$$

$$-\frac{d}{dx} \left(k A \frac{\partial u}{\partial t} \right) - \lambda \rho c A U = 0 \quad (\text{p-2})$$

The negative sign of the constant λ is based on the physical requirement that the solution $U(x)$ be **harmonic in x** while $T(t)$ decay **exponentially** with **increasing t**

The solution of (p-1) is

$$T(t) = K e^{-\lambda t}$$

Eigenvalue problems

$$T(t) = Ke^{-\lambda t}$$

where k is a constant of integration

- The values of λ are determined by solving (p-2), which also gives $U(x)$
- With $T(t)$ and $U(x)$ known, we have the complete homogeneous solution
- The problem of solving (p-2) for λ and $U(x)$ is termed an eigenvalue problem, and λ is called the eigenvalue and $U(x)$ the eigenfunction

When $K, A, \rho,$ and c are constants, the solution of (p-2) is

$$U(x) = C \sin \alpha x + D \cos \alpha x, \alpha^2 = \frac{\rho c}{k} \lambda \quad (\text{p-3})$$

where C and D are constants of integration. **Boundary conditions** of the problem **are used** to find algebraic relations among C and D

Eigenvalue problems

To fix the ideas, consider Eq. (p-2) subject to the boundary conditions: (e. g, a fin with specified temperature at $x = 0$ and insulated at $x = L$)

$$U(0) = 0, \left[kA \frac{dU}{dx} \right]_{x=L} = 0$$

Using the above boundary conditions in (p-2), we obtain

$$0 = C \cdot 0 + D \cdot 1, 0 = \alpha(C \cos \alpha L - D \sin \alpha L)$$



$$\left(\begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} - \alpha \begin{bmatrix} 0 & 0 \\ \cos \alpha L & \sin \alpha L \end{bmatrix} \right) \begin{Bmatrix} C \\ D \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} \quad (\text{p-4})$$

For nontrivial solution (i. e, not both C and D are equal to zero) , we set the **determinant** of the coefficient matrix in (p-4) to zero and obtain (since α cannot be zero)

$$\cos \alpha L = 0 \rightarrow \alpha_n L = \frac{(2n - 1)\pi}{2}$$

Eigenvalue problems

Hence, the homogeneous solution becomes

$$u^h(x, t) = \sum_{n=1}^{\infty} C_n e^{-\lambda_n t} \sin \alpha_n x, \lambda_n = \alpha_n^2 \left(\frac{k}{\rho c} \right), \alpha_n = \frac{(2n-1)\pi}{2L}$$

The constants C_n are determined using the initial condition of the problem, $u(x, 0) = u_0(x)$

$$u^h(x, 0) = \sum_{n=1}^{\infty} C_n \sin \alpha_n x = u_0(x)$$

Multiplying both sides with $\sin \alpha_m x$, integrating over the interval $(0, L)$, and making use of the orthogonality condition

$$\int_0^L \sin \alpha_n x \sin \alpha_m x dx = \begin{cases} 0, & \text{if } m \neq n \\ \frac{L}{2}, & \text{if } m = n \end{cases} \quad C_n = \frac{2}{L} \int_0^L u_0(x) \sin \alpha_n x dx$$

The complete solution of the **Parabolic equation** is given by the sum of the homogeneous solution and the particular solution

$$u(x, t) = u^h(x, t) + u^p(x, t)$$

Eigenvalue problems

Hyperbolic Equation

The axial motion of a bar, for example, is described by the equation

$$\rho A \frac{\partial^2 u}{\partial t^2} - \frac{\partial}{\partial x} \left(EA \frac{\partial u}{\partial x} \right) = f(x, t)$$

where u denotes the axial displacement, E the modulus of elasticity, A the cross-sectional area, ρ the density, and f the axial force per unit length

- The solution consists of two parts: **homogeneous** solution u^h (i.e, when $f = 0$) and **particular** solution up. The homogeneous part is determined by the separation-of-variables technique, as we discussed for the parabolic equation

The homogenous solution is also assumed to be of the form:

$$u^h(x, t) = U(x)T(t)$$

Substitution into the homogeneous form of hyperbolic eq., gives

Eigenvalue problems

$$\rho A \frac{d^2 T}{dt^2} - \frac{d}{dx} \left(EA \frac{dU}{dx} \right) T = 0$$

Assuming that ρA and EA are functions of x only, we arrive at

$$\frac{1}{T} \frac{d^2 T}{dt^2} = \frac{1}{\rho A U} \frac{d}{dx} \left(kA \frac{dU}{dx} \right) = -\alpha^2$$

or

$$\frac{d^2 T}{dt^2} + \alpha^2 T = 0 \quad \text{(h-1)}$$

$$\frac{d}{dx} \left(EA \frac{dU}{dx} \right) - \alpha^2 \rho A U = 0 \quad \text{(h-2)}$$

The solution of (h-1) is

$$T(t) = K e^{-i\alpha t} = K_1 \cos \alpha t + K_2 \sin \alpha t$$

The solution of (h-2) is

$$U(x) = C \sin \bar{\alpha} x + D \cos \bar{\alpha} x, \quad \bar{\alpha}^2 = \frac{\rho}{E} \alpha^2$$

the constants C and D are determined using the **boundary conditions** of the problem

Eigenvalue problems

Once again, we are required to solve an eigenvalue problem (the steps are **analogous to** those described for a **parabolic equation**)

Alternatively,

Eigenvalue problems associated with **parabolic** equations are obtained from corresponding equations of motion by assuming solution of the form:

$$u(x, t) = U(x)e^{-\alpha t}, \lambda = \alpha$$

Eigenvalue problems associated with **hyperbolic** equations are obtained by assuming solution of the form:

$$u(x, t) = U(x)e^{-i\omega t}, \lambda = \omega^2$$

λ denotes the eigenvalue

Also (p-2), (h-2) will be derived

Eigenvalue problems

Finite Element Formulation

Comparison of **parabolic** and **hyperbolic** eqs. with the previous model equation reveals that the equations governing eigenvalue problems are special cases of the model equations studied

Parabolic eq.

$$\frac{dT}{dt} = -\lambda T \quad (\text{p-1})$$

$$-\frac{d}{dx} \left(kA \frac{dU}{dx} \right) - \lambda \rho c AU = 0 \quad (\text{p-2})$$

Hyperbolic eq.

$$\frac{d^2 T}{dt^2} + \alpha^2 T = 0 \quad (\text{h-1})$$

$$-\frac{d}{dx} \left(EA \frac{dU}{dx} \right) - \alpha^2 \rho AU = 0 \quad (\text{h-2})$$

Second order differential eq. in Class-3

$$-\frac{d}{dx} \left(a \frac{du}{dx} \right) + cu - f = 0 \quad \text{for } 0 < x < L$$

Eigenvalue problems

Here we will summarize the steps in the finite element formulation of eigenvalue problems for the sake of completeness

- We will consider eigenvalue problems described by
 1. A single equation in a single unknown
(e. g, heat transfer, bar, and Euler-Bernoulli beam problems)
 2. A pair of equations in two variables
(e.g, Timoshenko beam theory)

Eigenvalue problems

Heat Transfer and Bar-Like Problems

Consider the problem of solving the equation

$$-\frac{d}{dx} \left[a(x) \frac{dU}{dx} \right] + c(x)U(x) = \lambda c_0(x)U(x)$$

for λ and $U(x)$. Here a , c , and c_0 are known quantities that depend on the physical problem, λ is eigenvalue, and U is eigenfunction. Special cases of above Eq. are given below

$$\text{Heat transfer: } a = kA, c = P\beta, c_0 = \rho cA$$

$$\text{Bars: } a = EA, c = 0, c_0 = \rho A$$

Over typical element Ω_e , we seek a finite element approximation of U in the form

$$U_h^e(x) = \sum_{j=1}^n u_j^e \psi_j^e(x)$$

Eigenvalue problems

The weak form of governing equation is

$$0 = \int_{x_a}^{x_b} \left(a \frac{dw}{dx} \frac{dU}{dx} + cwU(x) - \lambda c_0 wU \right) dx - Q_1^e w(x_a) - Q_n^e w(x_b)$$

where w is the weight function, and Q_1^e and Q_n^e are the secondary variables at node 1 and node n , respectively

$$Q_1^e = - \left[a \frac{dU}{dx} \right]_{x_a}, \quad Q_n^e = - \left[a \frac{dU}{dx} \right]_{x_b}$$

Substitution of the finite element approximation into the weak form gives the finite element model of the eigenvalue equation

$$[K^e]\{u^e\} - \lambda[M^e]\{u^e\} = \{Q^e\}$$

$$K_{i,j}^e = \int_{x_a}^{x_b} \left[a(x) \frac{d\psi_i^e}{dx} \frac{d\psi_j^e}{dx} + c(x) \psi_i^e \psi_j^e \right] dx, \quad M_{i,j}^e = \int_{x_a}^{x_b} c_0(x) \psi_i^e \psi_j^e dx$$

Above equation contains the finite element models of the eigenvalue equations (p-2) and (h-2) as **special cases**

Eigenvalue problems

Example

Consider a plane wall, initially at a uniform temperature T_0 , which has both surfaces suddenly exposed to a fluid at temperature T_∞ . The governing differential equation is

$$k \frac{\partial^2 T}{\partial x^2} = \rho c_0 \frac{\partial T}{\partial t}$$

and the initial condition is

$$T(x, 0) = T_0$$

where k is the thermal conductivity, ρ the density, and c_0 the specific heat at constant pressure, Equation is also known as the **diffusion equation** with diffusion coefficient $a = k/\rho c_0$

- We consider two sets of boundary conditions, each being representative of a different scenario for $x = L$. It amounts to solving for two different sets of boundary conditions

Eigenvalue problems

BC Set 1: If the heat transfer coefficient at the surfaces of the wall is assumed to be infinite the boundary conditions can be expressed as

$$T(0, t) = T_\infty, T(L, t) = T_\infty \text{ for } t > 0$$

BC Set 2: If we assume that the wall at $x = L$ is subjected to ambient temperature, we have

$$T(0, t) = T_\infty, \left[k \frac{\partial T}{\partial x} + \beta(T - T_\infty) \right] \Big|_{x=L} = 0$$

Equation can be normalized to make the boundary conditions homogeneous. Let

$$\alpha = \frac{k}{\rho c_0}, \hat{x} = \frac{x}{L}, \hat{t} = \frac{\alpha t}{L^2}, u = \frac{T - T_\infty}{T_0 - T_\infty}$$

The differential equation, initial condition, and boundary conditions become

Eigenvalue problems

$$-\frac{\partial^2 u}{\partial x^2} + \frac{\partial u}{\partial t} = 0$$

$$u(0, t) = 0, u(1, t) = 0, u(x, 0) = 1 \quad BC \text{ set 1}$$

$$u(0, t) = 0, \left. \left(\frac{\partial u}{\partial x} + Hu \right) \right|_{x=1} = 0, H = \frac{\beta L}{k} \quad BC \text{ set 2}$$

where the bars over x and u are omitted in the interest of brevity

By **separation-of variables technique** (or substitute $u = Ue^{-\lambda t}$) leads to the solution of the eigenvalue problem:

$$-\frac{d^2 U}{dx^2} - \lambda U = 0, U(0) = 0, U(1) = 0 \quad BC \text{ set 1}$$

$$-\frac{d^2 U}{dx^2} - \lambda U = 0, U(0) = 0, \left. \left(\frac{dU}{dx} + HU \right) \right|_{x=1} = 0 \quad BC \text{ set 2}$$

Eigenvalue problems

$$-\frac{d^2U}{dx^2} - \lambda U = 0$$

Recall in Heat Transfer and Bar-Like Problems, we solve

$$-\frac{d}{dx} \left[a(x) \frac{dU}{dx} \right] + c(x)U(x) = \lambda c_0(x)U(x)$$

This differential equation is a special case with $a = 1$, $c = 0$, and $c_0 = 1$. For a **linearelement**, the element equations have the explicit form:

$$\left(\frac{1}{h_c} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} - \lambda \frac{h_c}{6} \begin{bmatrix} 2 & 1 \\ 1 & 2 \end{bmatrix} \right) \begin{Bmatrix} u_1^e \\ u_2^e \end{Bmatrix} = \begin{Bmatrix} Q_1^e \\ Q_2^e \end{Bmatrix}$$

For a quadratic element, we have

$$\left(\frac{1}{3h_c} \begin{bmatrix} 7 & -8 & 1 \\ -8 & 16 & -8 \\ 1 & -8 & 7 \end{bmatrix} - \lambda \frac{h_e}{30} \begin{bmatrix} 4 & 2 & -1 \\ 2 & 16 & 2 \\ -1 & 2 & 4 \end{bmatrix} \right) \begin{Bmatrix} u_1^e \\ u_2^e \\ u_3^e \end{Bmatrix} = \begin{Bmatrix} Q_1^e \\ Q_2^e \\ Q_3^e \end{Bmatrix}$$

Eigenvalue problems

Solution for Set 1

For a mesh of **two linear elements** (the minimum number needed for Set 1 boundary conditions), with $h_1 = h_2 = 0.5$, the assembled equations are

$$\left(2 \begin{bmatrix} 1 & -1 & 0 \\ -1 & 2 & -1 \\ 0 & -1 & 1 \end{bmatrix} - \lambda \frac{1}{12} \begin{bmatrix} 2 & 1 & 0 \\ 1 & 4 & 1 \\ 0 & 1 & 2 \end{bmatrix} \right) \begin{Bmatrix} U_1 \\ U_2 \\ U_3 \end{Bmatrix} = \begin{Bmatrix} Q_1^1 \\ Q_2^1 + Q_1^2 \\ Q_2^2 \end{Bmatrix}$$

The boundary conditions $U(0) = 0$, $Q_2^1 + Q_1^2 = 0$, and $U(1) = 0$ require $U_1 = U_3 = 0$. Hence, the eigenvalue problem reduces to the single equation

$$\left(4 - \lambda \frac{4}{12} \right) U_2 = 0, \text{ or } \lambda_1 = 12.0, U_2 \neq 0$$

The mode shape is given (within an arbitrary constant take $U_2 = 1$) by

$$U(x) = U_2 \Phi_2(x) = \begin{cases} U_2 \psi_2^1(x) = x/h = 2x & 0 \leq x \leq 0.5 \\ U_2 \psi_2^1(x) = (2h - x)/h = 2(1 - x) & 0.5 \leq x \leq 1.0 \end{cases}$$

Eigenvalue problems

For a mesh of one quadratic element, we have ($h = 1.0$)

$$\frac{16}{3} - \lambda \frac{16}{30} = 0, \text{ or } \lambda_1 = 10.0, U_2 \neq 0$$

The corresponding eigenfunction is

$$U(x) = U_2 \Phi_2(x) = U_2 \psi_2^1 = 4 \frac{x}{h} \left(1 - \frac{x}{h} \right), 0 \leq x \leq 1.0$$

The exact eigenvalues: $\lambda_n = (n\pi)^2$ and $\lambda_1 = (\pi)^2 = 9.8696$

By comparison, **one quadratic element gives more accurate solution than two linear elements**

Eigenvalue problems

Natural Vibration of Beams

Euler-Bernoulli Beam

For the Euler-Bernoulli beam theory, the equation of motion is of the form:

$$\rho A \frac{\partial^2 w}{\partial t^2} - \rho I \frac{\partial^4 w}{\partial t^2 \partial x^2} + \frac{\partial^2}{\partial t^2} \left(EI \frac{\partial^2 w}{\partial t^2} \right) = q(x, t)$$

where ρ denotes the mass density per unit length, A the area of cross section, E the modulus and I the second moment of area. The expression involving ρI is called **rotary inertia term**

- Equation can be formulated as an eigenvalue problem in the interest of finding the **frequency of natural vibration** by assuming periodic motion

$$w(x, t) = W(x)e^{-i\omega t}$$

where ω is the **frequency** of natural transverse motion and $W(x)$ is the **mode shape** of the transverse motion. Substitution into Euler-Bernoulli beam equation, yields

Eigenvalue problems

$$\frac{d^2}{dx^2} \left(EI \frac{d^2 w}{dx^2} \right) - \lambda \left(\rho A W - \rho I \frac{d^2 w}{dx^2} \right) = 0 \quad \lambda = w^2$$

The weak form of above Eq. is given by

$$0 = \int_{x_a}^{x_b} \left(EI \frac{d^2 v}{dx^2} \frac{d^2 W}{dx^2} - \lambda \rho A v W - \lambda \rho I \frac{dv}{dx} \frac{dW}{dx} \right) dx$$
$$+ \left\{ v \left[\frac{d}{dx} \left(EI \frac{d^2 W}{dx^2} \right) + \lambda \rho I \frac{dW}{dx} \right] \right\}_{x_a}^{x_b} + \left[\left(-\frac{dv}{dx} \right) EI \frac{d^2 W}{dx^2} \right]_{x_a}^{x_b}$$

where v is the weight function

Note: The rotary inertia term contributes to the shear force term, giving rise to an effective shear force that must be known at a boundary point when the deflection is unknown at the point

To obtain the finite element model of weak form, assume finite element approximation of the form

$$W(x) = \sum_{j=1}^4 \Delta_j^e \phi_j^e(x)$$

where ϕ_j^e are the **Hermite cubic** polynomials

Eigenvalue problems

We obtain the finite element model

$$([K^e] - w^2[M^e])\{\Delta^e\} = \{Q^e\}$$

$$K_{ij}^e = \int_{x_a}^{x_b} EI \frac{d^2 \phi_i^e}{dx^2} \frac{d^2 \phi_j^e}{dx^2} dx, \quad M_{ij}^e = \int_{x_a}^{x_b} \left(\rho A \phi_i^e \phi_j^e + \rho I \frac{d\phi_i^e}{dx} \frac{d\phi_j^e}{dx} \right) dx$$

$$Q_1^e = \left[\frac{d}{dx} \left(EI \frac{d^2 W}{dx^2} \right) + \lambda \rho I \frac{dW}{dx} \right] \Big|_{x_a}, \quad Q_2^e = \left(EI \frac{d^2 W}{dx^2} \right) \Big|_{x_a}$$

$$Q_3^e = - \left[\frac{d}{dx} \left(EI \frac{d^2 W}{dx^2} \right) + \lambda \rho I \frac{dW}{dx} \right] \Big|_{x_b}, \quad Q_4^e = - \left(EI \frac{d^2 W}{dx^2} \right) \Big|_{x_b}$$

For constant values of EI and ρA , the **stiffness matrix** $[K^e]$ and **mass matrix** $[M^e]$ are

$$[K^e] = \frac{2E_e I_e}{h_e^3} \begin{bmatrix} 6 & -3h_e & -6 & -3h_e \\ -3h_e & 2h_e^2 & 3h_e & h_e^2 \\ -6 & 3h_e & 6 & 3h_e \\ -3h_e & h_e^2 & 3h_e & 2h_e^2 \end{bmatrix}$$

Eigenvalue problems

$$[M^e] = \frac{\rho^e A_e h_e}{420} \begin{bmatrix} 156 & -22h_e & 54 & 13h_e \\ -22h_e & 4h_e^2 & -13h_e & -3h_e^2 \\ 54 & -13h_e & 156 & 22h_e \\ 13h_e & -3h_e^2 & 22h_e & 4h_e^2 \end{bmatrix} + \frac{\rho^e I_e}{30h_e} \begin{bmatrix} 36 & -3h_e & -36 & -3h_e \\ -3h_e & 4h_e^2 & 3h_e & -h_e^2 \\ -36 & 3h_e & 36 & 3h_e \\ -3h_e & -h_e^2 & 3h_e & 4h_e^2 \end{bmatrix}$$

When rotary inertia is neglected, we omit the second part of the mass matrix in $[M^e]$

Eigenvalue problems

Timoshenko Beam

For the Timoshenko beam theory, the equation of motion is of the form:

$$\begin{aligned}\rho A \frac{\partial^2 w}{\partial t^2} - \frac{\partial}{\partial x} \left[GAK_s \left(\frac{\partial w}{\partial x} + \Psi \right) \right] &= 0 \\ \rho I \frac{\partial^2 \Psi}{\partial t^2} - \frac{\partial}{\partial x} \left(EI \frac{\partial \Psi}{\partial x} \right) + GAK_s \left(\frac{\partial w}{\partial x} + \Psi \right) &= 0\end{aligned}$$

where G is the shear modulus ($G = E/2(1 + \nu)$) and K_s is the shear correction factor ($K_s = 5/6$). Note that above Eq. contains the rotary inertia term. Once again, we assume periodic motion and write

$$w(x, t) = W(x)e^{-i\omega t}, \Psi(x, t) = S(x)e^{-i\omega t}$$

and obtain the eigenvalue problem

$$\begin{aligned}-\frac{d}{dx} \left[GAK_s \left(\frac{\partial w}{\partial x} + s \right) \right] - w^2 \rho A W &= 0 \\ -\frac{d}{dx} \left(EI \frac{ds}{dx} \right) + GAK_s \left(\frac{\partial w}{\partial x} + s \right) - w^2 \rho I S &= 0\end{aligned}$$

Eigenvalue problems

For equal interpolation of $W(x)$ and $S(x)$,

$$W(x) = \sum_{j=1}^n W_j^e \psi_j^e(x), S(x) = \sum_{j=1}^n S_j^e \psi_j^e(x)$$

where ψ_j^e are the $(n - 1)$ order Lagrange polynomials, the finite element model is given by

$$\left(\begin{bmatrix} [K^{11}] & [K^{12}] \\ [K^{21}] & [K^{22}] \end{bmatrix} - w^2 \begin{bmatrix} [M^{11}] & 0 \\ 0 & [M^{22}] \end{bmatrix} \right) \begin{Bmatrix} \{W\} \\ \{S\} \end{Bmatrix} = \begin{Bmatrix} \{F^1\} \\ \{F^2\} \end{Bmatrix}$$

$[K^e]$ is the stiffness matrix and $[M^e]$ is the mass matrix

$$K_{ij}^{11} = \int_{x_a}^{x_b} GAK_s \frac{d\psi_i^e}{dx} \frac{d\psi_j^e}{dx} dx$$

$$K_{ij}^{12} = \int_{x_a}^{x_b} GAK_s \frac{d\psi_i^e}{dx} \psi_j^e dx = K_{ji}^{21}$$

$$K_{ij}^{22} = \int_{x_a}^{x_b} \left(EI \frac{d\psi_i^e}{dx} \frac{d\psi_j^e}{dx} + GAK_s \psi_i^e \psi_j^e \right) dx,$$

$$M_{ij}^{11} = \int_{x_a}^{x_b} \rho A \psi_i^e \psi_j^e dx, M_{ij}^{22} = \int_{x_a}^{x_b} \rho AI \psi_i^e \psi_j^e dx,$$

Eigenvalue problems

$$F_j^1 = Q_{2i-1}, F_j^2 = Q_{2i}$$

$$Q_1^e \equiv - \left[GAK_s \left(s + \frac{dW}{dx} \right) \right] \Big|_{x_a}, Q_2^e \equiv - \left(EI \frac{dS}{dx} \right) \Big|_{x_a}$$

$$Q_3^e \equiv \left[GAK_s \left(s + \frac{dW}{dx} \right) \right] \Big|_{x_b}, Q_4^e \equiv - \left(EI \frac{dS}{dx} \right) \Big|_{x_b}$$

For the choice of **linear** interpolation functions, we have

$$[K^e] = \left(\frac{2E_e I_e}{\mu_0 h_e^3} \right) \begin{bmatrix} 6 & -3h_e & -6 & -3h_e \\ -3h_e & h_e^2(1.5 + 6\Lambda_e) & 3h_e & h_e^2(1.5 - 6\Lambda_e) \\ -6 & 3h_e & 6 & 3h_e \\ -3h_e & h_e^2(1.5 - 6\Lambda_e) & 3h_e & h_e^2(1.5 + 6\Lambda_e) \end{bmatrix}$$

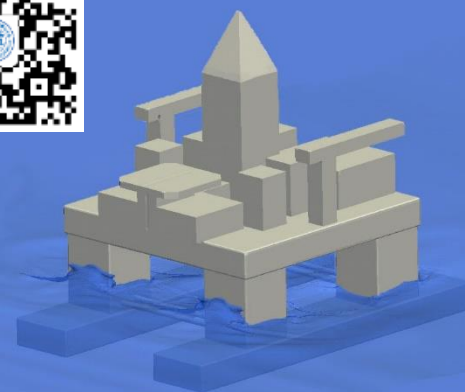
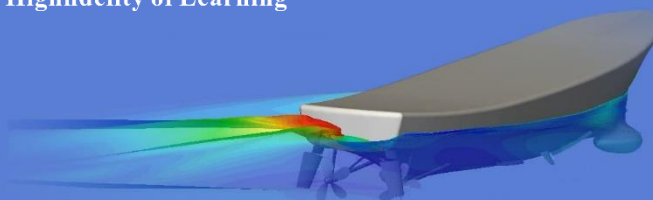
$$[M^e] = \frac{\rho^e A_e}{6} \begin{bmatrix} 2 & 0 & 1 & 0 \\ 0 & 2r_e & 0 & r_e \\ 1 & 0 & 2 & 0 \\ 0 & r_e & 0 & 2r_e \end{bmatrix}, r_e = \frac{I_e}{A_e}$$

$$\Lambda_e = \frac{E_e I_e}{G_e A_e K_s h_e^2}, \mu_0 = 12\Lambda_e$$

谢谢!

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